

Fundamentals of Drafting - Tolerances

Objectives:



1. To appreciate the need for tolerances in terms of
 - (a) function
 - (b) interchangeability
 - (c) cost
2. To define the principal terms of limits and fits as specified in BS 4500 Part 1 and to sketch and label diagrams to illustrate them.
3. To differentiate between hole-basis fits and shaft-basis fits.
4. To calculate the maximum and minimum limits for shaft and holes given nominal sizes and classes of fits.

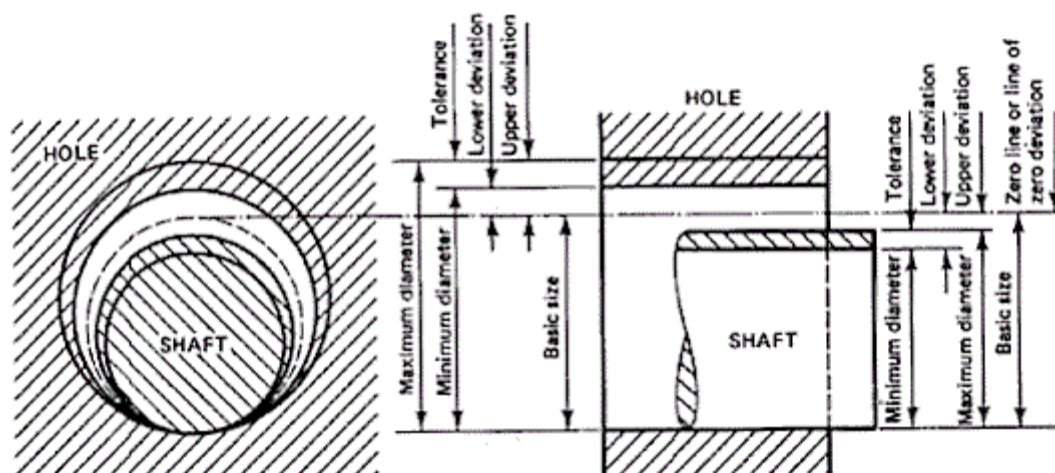
Need for tolerances

Tolerance is the amount of deviation from the given basic design size, or the margin of error, allowable to accommodate reasonable inaccuracy in manufacture. In mechanical drawings, tolerance is indicated by the *maximum and minimum permitted sizes* (limits of sizes).

There are three main considerations in establishing reasonable tolerances.

- (a) **Function** setting the limits of dimensions closely enough to ensure that the component will perform its function satisfactorily at the extent of the limits, i.e. **tolerance must be consistent with the design and function**
- (b) **Interchangeability** setting the limits of dimensions for components so that a component taken at random from a batch should be able to be assembled without modification, i.e. **tolerance should be specified for all dimensions affecting interchangeability**
- (c) **Cost** setting the limits of dimensions at no more closely than they need to be in order to minimise the production cost, i.e. **tolerance must be as large as the design will permit**

Principal terms expressing limits of size

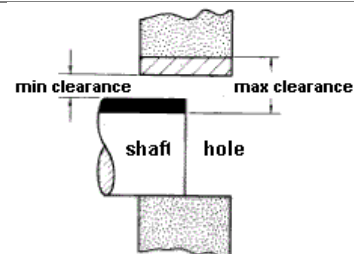


<u>Size</u>	Size is a number expressing in a particular unit the numerical value of a length.
<u>Actual size</u>	Actual size is the size of a part as obtained by measurement.
<u>Basic size</u>	Basic size is the size from which the limits are fixed. It is the same for both members of a fit.
<u>Limits of size</u>	Limits of size are the maximum and minimum sizes permitted for a feature.
<u>Maximum limit of size</u>	Maximum limit of size is the maximum size permitted for a feature.
<u>Minimum limit of size</u>	Minimum limit of size is the minimum size permitted for a feature.
<u>Deviation</u>	Deviation is the algebraic difference between the actual size and the corresponding basic size.
<u>Upper deviation</u>	Upper deviation is the algebraic difference between the maximum limit of size and the corresponding basic size.
<u>Lower deviation</u>	Lower deviation is the algebraic difference between the minimum limit of size and the corresponding basic size.
<u>Zero line</u>	Zero line is the line of zero deviation and it represents the basic size.
<u>Shaft</u>	Shaft is a term used to designate all external features of a part, including those which are not cylindrical.
<u>Hole</u>	Hole is a term used to designate all internal features of a part, including those which are not cylindrical.

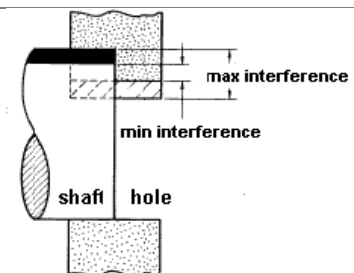
Systems of fits

A fit is the relationship resulting from the difference, before assembly, between the sizes of the two parts which are to be assembled. Engineering fits can be divided into three main types:

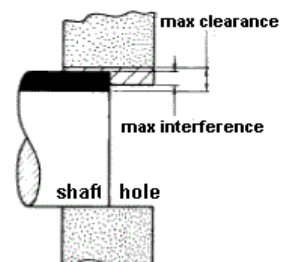
- (a) Clearance fit
 a fit which always provides a clearance
 e.g. a shaft is smaller than the hole into which it fits



- (b) Interference fit
 a fit which always provides an interference
 e.g. a shaft is bigger than the hole into which it fits



- (c) Transition fit
 a fit which may provide either a clearance or an interference
 e.g. a shaft is bigger, smaller or equal to the hole into which it fits



A system of fits comprises shafts and holes which belong to a limit system. There are two systems of fits:

- (a) **Hole-basis fits** this system of fits is based on a single hole, from which the different clearances and interferences are obtained by associating various shafts with the hole
- (b) **Shaft-basis fits** this system of fits is based on a single shaft, with which the different clearances and interferences are obtained by associating various holes with the shaft

The hole-basis system of fits is more economical than the shaft-basis system as only one size of drill is used to produce different fits, the shafts being turned and ground to the required sizes, thus making manufacture and measurement much easier.

Selected fits

BS 4500 Part 1 provides a selection of hole and shaft tolerances to cover a wide range of engineering applications. The following selected hole and shaft tolerances have been found to be commonly applied:

- (a) Selected hole tolerances H7; H8; H9; H11
- (b) Selected shaft tolerances c11; d10; e9; f7; g6; h6; k6; n6; p6; s6

Tolerance is designated by a letter and a figure. A hole tolerance is designed by a capital letter indicating the fundamental deviation, following by a figure denoting the grade (e.g. H7). Shaft fundamental deviations are indicated by small letters, followed by a tolerance grade (e.g. p6). The hole limits are always quoted first, e.g. H7/p6 stands for a press fit.

Data sheets showing a selected range of fits can be derived from the hole tolerances and shaft tolerances. Data sheet BS 4500A gives a range of hole-basis fits derived from the selected hole and shaft tolerances given in BS 4500 Part 1. It covers fits from loose clearance to heavy interference and is suitable for most normal requirements.

Data sheet BS 4500B shows the shaft-basis equivalents of the hole-basis fits in BS 4500A.

BRITISH STANDARD
SELECTED ISO FITS—HOLE BASIS

Data Sheet
 4500A
 Issue 2 February 1979

Refer to your text on page 65

Nominal size	Tolerance	Tolerances										Nominal size	Tolerance
		H11	H10	H9	H8	H7	h11	h10	h9	h8	h7		
3	+0.050	-0.015	-0.025	-0.035	-0.045	-0.055	-0.065	-0.075	-0.085	-0.095	-0.105	3	-0.015
6	+0.075	-0.020	-0.030	-0.040	-0.050	-0.060	-0.070	-0.080	-0.090	-0.100	-0.110	6	-0.020
10	+0.100	-0.025	-0.035	-0.045	-0.055	-0.065	-0.075	-0.085	-0.095	-0.105	-0.115	10	-0.025
18	+0.150	-0.030	-0.040	-0.050	-0.060	-0.070	-0.080	-0.090	-0.100	-0.110	-0.120	18	-0.030
30	+0.200	-0.035	-0.045	-0.055	-0.065	-0.075	-0.085	-0.095	-0.105	-0.115	-0.125	30	-0.035
40	+0.250	-0.040	-0.050	-0.060	-0.070	-0.080	-0.090	-0.100	-0.110	-0.120	-0.130	40	-0.040
50	+0.300	-0.045	-0.055	-0.065	-0.075	-0.085	-0.095	-0.105	-0.115	-0.125	-0.135	50	-0.045
65	+0.350	-0.050	-0.060	-0.070	-0.080	-0.090	-0.100	-0.110	-0.120	-0.130	-0.140	65	-0.050
80	+0.400	-0.055	-0.065	-0.075	-0.085	-0.095	-0.105	-0.115	-0.125	-0.135	-0.145	80	-0.055
100	+0.450	-0.060	-0.070	-0.080	-0.090	-0.100	-0.110	-0.120	-0.130	-0.140	-0.150	100	-0.060
120	+0.500	-0.065	-0.075	-0.085	-0.095	-0.105	-0.115	-0.125	-0.135	-0.145	-0.155	120	-0.065
140	+0.550	-0.070	-0.080	-0.090	-0.100	-0.110	-0.120	-0.130	-0.140	-0.150	-0.160	140	-0.070
160	+0.600	-0.075	-0.085	-0.095	-0.105	-0.115	-0.125	-0.135	-0.145	-0.155	-0.165	160	-0.075
180	+0.650	-0.080	-0.090	-0.100	-0.110	-0.120	-0.130	-0.140	-0.150	-0.160	-0.170	180	-0.080
200	+0.700	-0.085	-0.095	-0.105	-0.115	-0.125	-0.135	-0.145	-0.155	-0.165	-0.175	200	-0.085
225	+0.750	-0.090	-0.100	-0.110	-0.120	-0.130	-0.140	-0.150	-0.160	-0.170	-0.180	225	-0.090
250	+0.800	-0.095	-0.105	-0.115	-0.125	-0.135	-0.145	-0.155	-0.165	-0.175	-0.185	250	-0.095
280	+0.850	-0.100	-0.110	-0.120	-0.130	-0.140	-0.150	-0.160	-0.170	-0.180	-0.190	280	-0.100
315	+0.900	-0.105	-0.115	-0.125	-0.135	-0.145	-0.155	-0.165	-0.175	-0.185	-0.195	315	-0.105
355	+0.950	-0.110	-0.120	-0.130	-0.140	-0.150	-0.160	-0.170	-0.180	-0.190	-0.200	355	-0.110
400	+1.000	-0.115	-0.125	-0.135	-0.145	-0.155	-0.165	-0.175	-0.185	-0.195	-0.205	400	-0.115
450	+1.050	-0.120	-0.130	-0.140	-0.150	-0.160	-0.170	-0.180	-0.190	-0.200	-0.210	450	-0.120

Use of data sheet

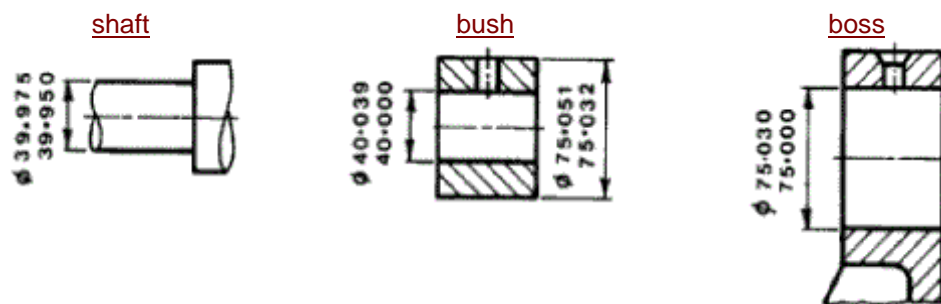
Data sheet can be used to determine the working limits of an assembly. The following example demonstrates how the limits of sizes are calculated using the data sheet BS 4500A.

The assembly	Sub-assembly	Desirable fit	Nominal dia
	shaft + bush	H8/f7 normal fit	40
	bush + boss	H7/p6 press fit	75

Extracting data from BS 4500A,

	Nominal size	Fit	Tolerance	Max limit	Min limit
shaft	40	f7	-25, -50	$40-0.025=39.975$	$40-0.05=39.950$
bush id	40	H8	+39, 0	$40+0.039=40.039$	$40+0=40.000$
bush od	75	p6	+51, +32	$75+0.051=75.051$	$75+0.032=75.032$
boss	75	H7	+30, 0	$75+0.030=75.030$	$75+0=75.000$

Dimensioning the parts,



Assignment 8

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Page 89

Question No. 35 (Referring to Figure 7.19)

The belt-pulley unit shown in Fig. 7.19 consists of a belt pulley (2), a mounting bracket (1), a fitted bolt (3), and two bushes (4). (The following figure is extracted from textbook.)

Draw full size in third angle projection with all parts assembled, including a suitable nut with locking device. Take all fillets to be R2

- (a) a sectional front view on YY,
- (b) a sectional end view on XX,
- (c) a plan view.

Insert a title block and add a parts list.

Hidden details are not required for all views

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Page 89

Question No. 36 (Referring to Figure 7.19)

Calculate the maximum and minimum limits for the following fits and add these dimensions on the assembly drawing above.

- (a) H7/p6 between shaft (3) and two bushes (4),
- (b) H9/e9 between two bushes (4) and belt pulley (2)

